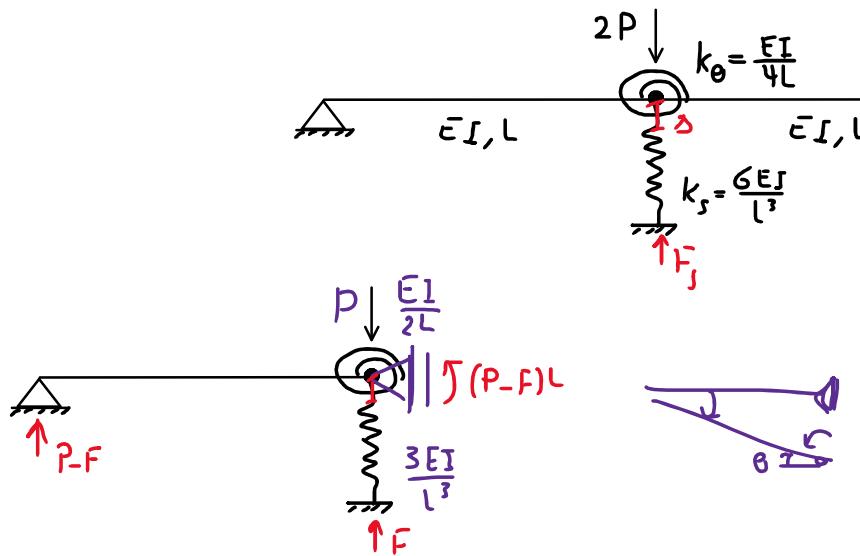


مثال: نیروی منزه را بدست آوردید.



$$F_s = k_s \Delta$$

$$F_{s/2} = \frac{k_s}{2} \Delta$$

$$M = k_\theta \Delta$$

$$M = 2k_\theta \Delta$$

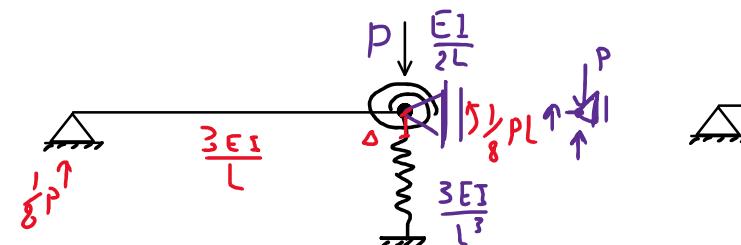
روش تیب - انت

$$M = \frac{3EI}{L} (\theta - \frac{\Delta}{L})$$

$$-(P-F)L = \frac{3EI}{L} \left(\frac{(P-F)L}{EI} - \frac{F}{2k} \right) \rightarrow F-P = 3 \left(2(P-F) - \frac{F}{3} \right)$$

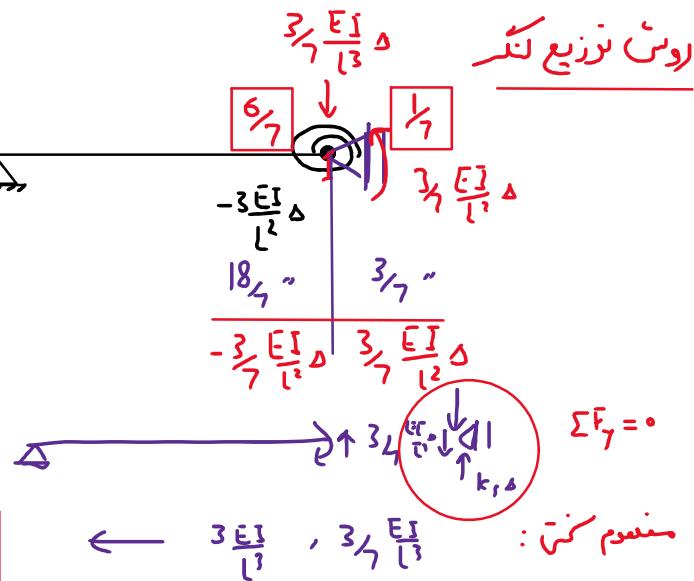
$$F-P = 8P - 7F$$

$$F_s = \frac{7}{8} P, \quad M_s = \frac{1}{8} PL$$

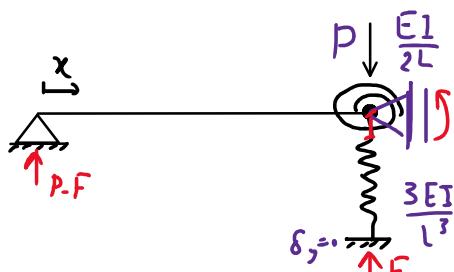


$$\frac{3}{7} \frac{EI}{L^3} \Delta + \frac{3}{7} \frac{EI}{L^3} \Delta = P \rightarrow \Delta = \frac{7}{24} \frac{PL^3}{EI}$$

$$F_s = \frac{3EI}{L^3} \times \frac{7}{24} \frac{PL^3}{EI} = \frac{7}{8} P$$



$$F_s = \frac{7}{8} P$$

منسجم کن: $\sum F_y = 0$ 

$$1/2 \int_0^L M^2 dx + 1/2 \frac{N^2}{k_s} + 1/2 \frac{M^2}{k_\theta}$$

$$\frac{\partial U}{\partial F} = \int \frac{M}{EI} \left(\frac{\partial M}{\partial F} \right) dx + \frac{N}{k_s} \left(\frac{\partial N}{\partial F} \right) + \frac{M}{k_\theta} \left(\frac{\partial M}{\partial F} \right) = 0$$

لوشی استیبلانو

$$U = \frac{1}{2} \int \frac{M^2}{EI} dx + \frac{1}{2} \frac{N^2}{k_s} + \frac{1}{2} \frac{M^2}{k_\theta}$$

$$\text{و} M = I \sigma z v, \quad \delta M = \dots$$

$$\delta_j = \frac{F}{k_s} \quad \text{of } J = EI \cdot \delta F$$

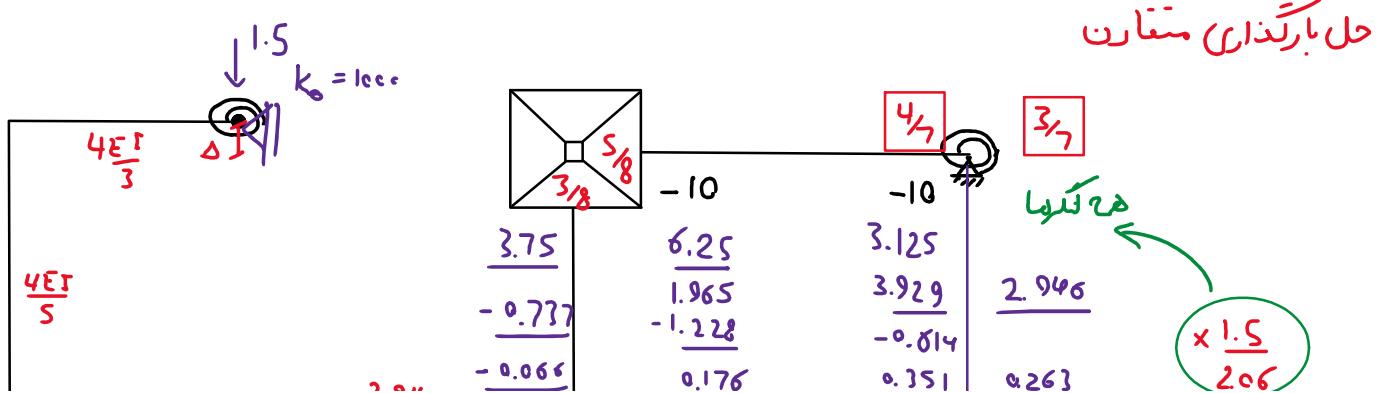
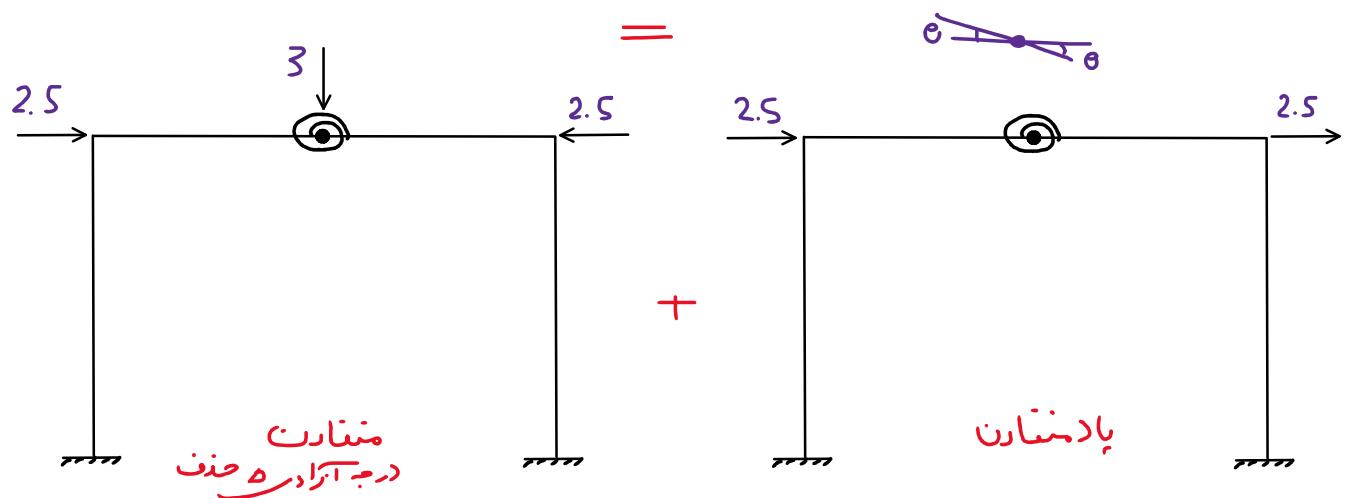
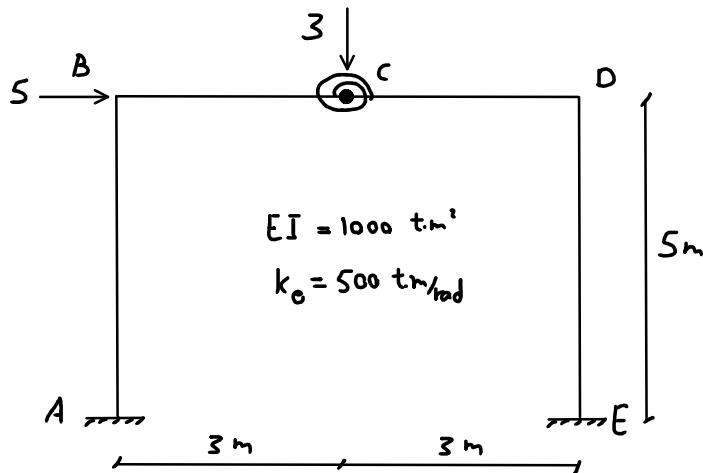
$$\frac{1}{EI} \int_0^L (F-P)x^2 dx + \frac{F}{k_s} + \frac{(F-P)L^2}{k_e} = 0$$

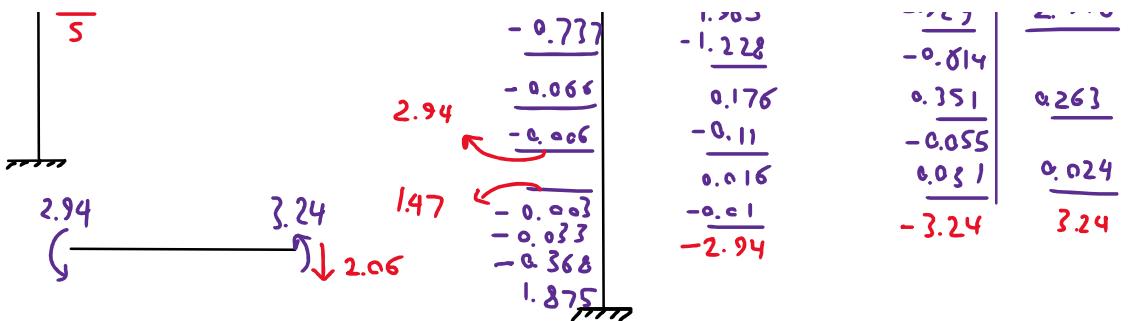
$$\frac{1}{EI} (F-P) \frac{x^3}{3} \Big|_0^L + \frac{FL^3}{3EI} + (F-P) \frac{2L^3}{EI} = 0$$

$$F = \frac{7}{8} P$$

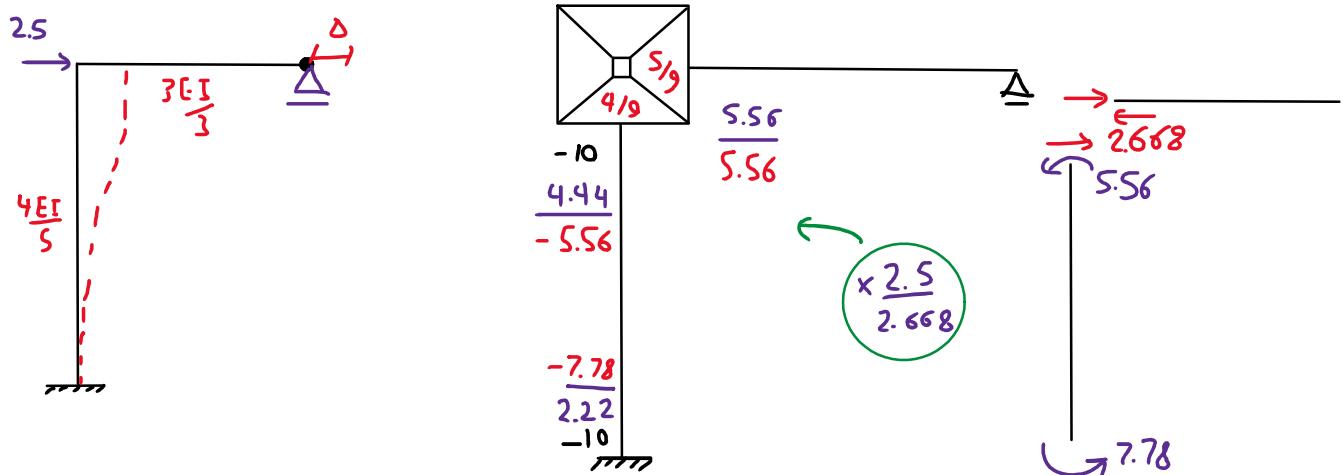
$K_s \text{ of } J = k_e \delta F$
 ① $M = (P-F)x , \frac{\partial M}{\partial F} = -x$
 ② $N = F , \frac{\partial N}{\partial F} = 1$
 ③ $M = (P-F)L , \frac{\partial M}{\partial F} = -L$

مثال: لذتماس انتقامی اعضا را به دست آورید.

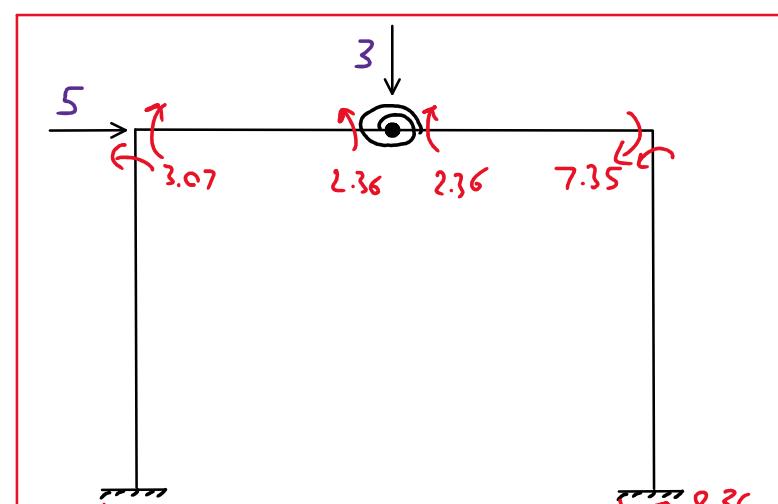
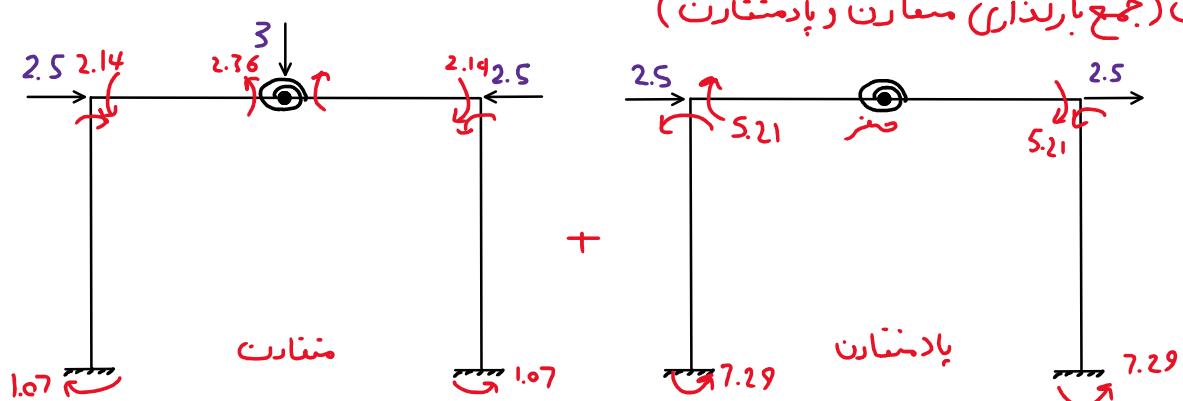




حل بارگذاری پادمترن



بارگذاری کل (جمع بارگذاری منوار و پادمترن)





ردش دیگر (ردش کاستیلیاف)

حل بارگذاری پادمترن

Diagram of a beam with a fixed support at 2.5 and a roller support at 7.29. A force F acts upwards at a distance of 5.21 from the fixed support. The beam has a length of 3.

$$\frac{\partial U}{\partial F} = \int \frac{M}{EI} \left(\frac{\partial M}{\partial F} \right) dx = 0$$

$$\begin{cases} ① M = Fx & \frac{\partial M}{\partial F} = x \\ ② M = 3F - 2.5x & \frac{\partial M}{\partial F} = 3 \end{cases}$$

$$\int_0^3 (Fx)(x) dx + \int_0^5 (3F - 2.5x)(3) dx = 0$$

$$F x \Big|_0^3 + (9F x - \frac{7.5}{2} x^2) \Big|_0^5 = 0 \rightarrow 9F + 45F - 97.75 = 0$$

$$F = 1.736$$

Diagram of a beam with a fixed support at 0 and a roller support at 7. The beam has a length of 3. A spring with stiffness $k_e = 1000$ is connected between the fixed support and a point at a distance of 1.5 from the fixed support. A force F acts downwards at this point. The beam has a constant flexural rigidity $EI = 1000$.

$$U = \frac{1}{2} \int \frac{M^2}{EI} dx + \frac{1}{2} \frac{M_e^2}{k_e}$$

$$\begin{cases} ① M = -M_e - 1.5x & \frac{\partial M}{\partial M_e} = -1 & \frac{\partial M}{\partial F} = 0 \\ ② M = -M_e - 4.5 + Fx & = -1 & = x \\ ③ M = M_e & = 1 & = 0 \end{cases}$$

$$\text{Eq. } ① \quad \frac{\partial U}{\partial M_e} = 0 \rightarrow \frac{1}{EI} \int_0^3 (M_e + 1.5x) dx + \frac{1}{k_e} \int_0^5 (M_e + 4.5 - Fx) dx + \frac{M_e}{k_e} = 0$$

$$(M_e x + 1.5 \frac{x^2}{2}) \Big|_0^3 + (M_e x + 4.5x - Fx^2) \Big|_0^5 + M_e = 0$$

$$3M_e + 6.75 + 5M_e + 22.5 - 12.5F + M_e = 0 \rightarrow * 9M_e - 12.5F = -29.25$$

Eq. ②

$$\frac{\partial U}{\partial F} = 0 \rightarrow 0 + \frac{1}{EI} \int_0^5 (-M_e x - 4.5x + Fx^2) dx + 0 = 0$$

$$(-M_e \frac{x^2}{2} - \frac{4.5}{2} x^2 + F \frac{x^3}{3}) \Big|_0^5 = 0 \rightarrow * -12.5M_e + 41.67F = 56.25$$

$$12.5 \begin{cases} 9M_e - 12.5F = -29.25 \\ -12.5M_e + 41.67F = 56.25 \end{cases}$$

$$218.75F = 140.625 \rightarrow$$

$$\boxed{F = 0.643}$$

$$\boxed{M_e = -2.36}$$

